

Fluid-driven fingering instability of a confined elastic meniscus

This content has been downloaded from IOPscience. Please scroll down to see the full text.

2015 EPL 110 34001

(<http://iopscience.iop.org/0295-5075/110/3/34001>)

View [the table of contents for this issue](#), or go to the [journal homepage](#) for more

Download details:

IP Address: 128.103.224.4

This content was downloaded on 04/08/2015 at 01:20

Please note that [terms and conditions apply](#).

Fluid-driven fingering instability of a confined elastic meniscus

JOHN S. BIGGINS¹, Z. WEI² and L. MAHADEVAN^{2,3(a)}

¹*Cavendish Laboratory, University of Cambridge - JJ Thomson Ave, Cambridge, CB3 0HE, UK*

²*School of Engineering and Applied Sciences, Harvard University - Cambridge, MA 02138, USA*

³*Department of Physics, Harvard University - Cambridge, MA 02138, USA*

received 3 December 2014; accepted in final form 15 April 2015

published online 15 May 2015

PACS 46.32.+x – Static buckling and instability

PACS 46.25.-y – Static elasticity

PACS 46.55.+d – Tribology and mechanical contacts

Abstract – When a fluid is pumped into a cavity in a confined elastic layer, at a critical pressure, destabilizing fingers of fluid invade the elastic solid along its meniscus (SAINTYVES B. *et al.*, *Phys. Rev. Lett.*, **111** (2013) 047801). These fingers occur without fracture or loss of adhesion and are reversible, disappearing when the pressure is decreased. We develop an asymptotic theory of pressurized highly elastic layers trapped between rigid bodies in both rectilinear and circular geometries, with predictions for the critical fluid pressure for fingering, and the finger wavelength. Our results are in good agreement with recent experimental observations of this elastic interfacial instability in a radial geometry. Our theory also shows that, perhaps surprisingly, this lateral-pressure-driven instability is analogous to a transverse-displacement-driven instability of the elastic layer. We verify these predictions by using non-linear finite-element simulations on the two systems which show that in both cases the fingering transition is first order (sudden) and hence has a region of bistability.

Copyright © EPLA, 2015

In continuum mechanics, fingering instabilities are usually associated with interfacial flow in porous media, or its analog, flow in a Hele-Shaw cell. Indeed, the prototypical Saffman-Taylor interfacial instability is associated with a less viscous fluid pushed into a more viscous liquid in a confined geometry, leading to the interface between the two developing finger-like protrusions [1,2], with a wavelength set by a balance between viscous stresses and surface tension [1]. The fingers show various morphologies as a function of the fluid properties, *e.g.* in the limit of high viscosity contrast and minimal surface tension, or when the fluid is viscoelastic, shear-thinning or viscoplastic, one sees a range of finger, fracture and fractal-like patterns [3–17]. Crucially, we note that these instabilities are all manifestations of fluid-like irreversible rate-dependent deformations.

Recently the solid/elastic analog of the Saffman-Taylor experiment has been explored by slowly pumping air into a cavity in a strongly adherent highly elastic layer in a Hele-Shaw cell [18]. The cavity first dilates laterally and at a critical pressure, fingers of air invaded the elastic layer without fracture or loss of adhesion, as shown in

fig. 1. When the pressure was released the cavity completely recovered its original shape, the hallmark of a purely elastic quasi-static transition. Two other fingering transitions have been reported in soft solid layers sitting betwixt rigid bodies which are then pulled apart. In one peeling causes adhesion between the layer and body to fail, and finger-like undulations appear along the resulting contact line [17,19,20]. In the other, adhesion is maintained and finger-like invaginations appear along the free boundary of the layer [21,22]. Both these latter instabilities involve control of the transverse displacement.

Here we provide a theory for the fluid-driven reversible elastic instability of a confined meniscus, and unify fingering in strongly adherent elastic layers produced by either pressure-driven fluid invasion or transverse displacement. Though familiar for viscous fingering, this unification is unexpected in elasticity where force and displacement loadings typically give different behaviors. While the fingering of polymeric fluids in the limit of high Deborah numbers (when the polymer cannot relax) [5,15–17] might seem to be similar to the rate-independent elastic deformations treated here, they are not since the former lead to irreversible deformations while the latter are completely reversible.

^(a)E-mail: lm@seas.harvard.edu

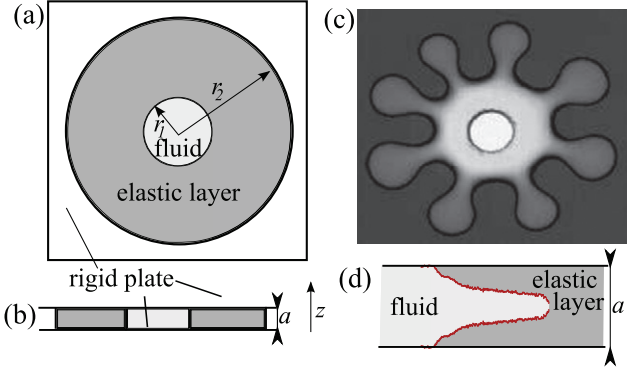


Fig. 1: (Colour on-line) (a) Top view of the experiment used to study the elastic analog of Saffman-Taylor fingering [18]: two rigid plates confine a thin elastic layer with a central cavity containing fluid whose volume is increased by injecting fluid from above. (b) Cross-section showing the thickness of the layer. (c) Experimentally obtained finger pattern [18]. The central hole indicates the original cavity size, while the varying gray scale is a consequence of the elastic meniscus deforming without loss of adhesion to the plates. (d) Cross-section of a finger [18] showing maintained adhesion even when the finger amplitude is large, well past the onset of the instability.

We begin with scaling estimates for fluid-driven elastic fingering in a thin incompressible neo-Hookean annular layer (fig. 1(a), (b)) adhered to rigid plates at $z = \pm a/2$ and with in-plane extent $r_1 < r < r_2$ ($r_1 \gg a$) and shear modulus μ . Since adhesion is maintained, an in-plane displacement u , applied mid-way between the plates, will generate strains $\gamma \sim u/a$ localized in plane by an elastic screening length of $\mathcal{O}(a)$, which we expect to be the finger wavelength scale. A fluid (pressure P_f) pumped into the cavity will induce such a displacement radially on the inner circumference, increasing its volume by $\delta V_f \sim 2\pi r_1 u a$. Since the layer is incompressible, this displacement decays radially as $u(r) \sim u r_1 / r$. Equating the layer elastic energy, $E \sim a \int_{r_1}^{r_2} \frac{1}{2} \mu \gamma^2 2\pi r dr \sim \mu u^2 r_1^2 \log(r_2/r_1)/a$, and the work done by the fluid, $P_f \delta V_f$, yields $u \sim (P_f/\mu) a^2 / (r_1 \log(r_2/r_1))$. Since fingering occurs when the strain is finite, *i.e.* $\gamma \sim u/a \gtrsim 1$, this yields a critical threshold pressure for the instability $P_f \sim \mu (r_1/a) \log(r_2/r_1)$.

To verify and improve these estimates, we build a minimal 2d theory, taking advantage of the scale separation induced by confinement. Consider a point with position $\mathbf{R} = \mathbf{r} + z\hat{\mathbf{z}}$, and displacement $\mathbf{V}(\mathbf{R}) = \mathbf{u}(\mathbf{R}) + v_\perp(\mathbf{R})\hat{\mathbf{z}}$, where \mathbf{r} and \mathbf{u} are in plane and $\hat{\mathbf{z}}$ is the layer normal. Expanding $\mathbf{V}(\mathbf{R})$ to second order in z , imposing symmetry around $z = 0$ and requiring $\mathbf{V}(\mathbf{R}) = 0$ at $z = \pm a/2$, we get an approximate form for the displacement¹,

$$\mathbf{V}(\mathbf{R}) = (1 - 2z/a)(1 + 2z/a)\mathbf{u}(\mathbf{r}), \quad (1)$$

reminiscent of the quadratic flow profile that underpins the theory of viscous fingering [1]. Soft incompressible

¹Figure 1(d) shows a non-quadratic profile, but the finger is in the non-linear regime well past the point of instability.

solids are well modeled by the neo-Hookean energy density $\frac{1}{2}\mu(\text{Tr}(F \cdot F^T) - 3)$, where $F_{\alpha\beta} = \delta_{\alpha\beta} + \partial_\beta \mathbf{V}_\alpha$, and incompressibility requires $\text{Det}(F) = 1$. Implementing incompressibility in a depth-averaged sense we define our 2d energy density by

$$L = \int_{-a/2}^{a/2} \frac{1}{2}\mu(\text{Tr}(F \cdot F^T) - 3) - P(\text{Det}(F) - 1) dz. \quad (2)$$

The quadratic form for \mathbf{V} gives $F = I + (1 - 4z^2/a)\nabla\mathbf{u}(\mathbf{r}) - 8z/a^2\mathbf{u}(\mathbf{r})\hat{\mathbf{z}} + \mathbf{z}\mathbf{z}$, where I and ∇ are the in-plane identity and gradient. Conducting the thickness (z) integral gives

$$L(\mathbf{u}, P) = \frac{5a}{6} \left(\frac{1}{2}\mu(\text{Tr}(G \cdot G^T) - 2) + \frac{16}{5}\mu \frac{\mathbf{u} \cdot \mathbf{u}}{a^2} - P(\text{Det}(G) - 1) \right), \quad (3)$$

where $G = I + \frac{4}{5}\nabla\mathbf{u}$ is an effective 2d deformation gradient, and P is a 2d pressure. Minimizing the elastic energy $E = \int L dA$ over \mathbf{u} and P leads to the Euler-Lagrange equations

$$\frac{8\mu}{a^2}\mathbf{u} = \frac{4\mu}{5}\nabla^2\mathbf{u} - \text{Det}(G)G^{-T} \cdot \nabla P, \quad (4)$$

$$\text{Det}(G) = 1. \quad (5)$$

To derive the associated boundary conditions, we imagine a small additional displacement $\delta\mathbf{u}$ that gives rise to a change in E arising at the boundary $\delta E = \frac{2a}{3} \oint \delta\mathbf{u} \cdot (\mu G - P \text{Det}(G)G^{-T}) \cdot \hat{\mathbf{n}} \cdot ds$, where $\hat{\mathbf{n}}$ is the boundary outward normal. At a free boundary δE would vanish. At an interface with fluid at pressure P_f we must add the virtual work term $-P_f \delta V_f$ (V_f is the fluid volume) to E , generating an additional boundary term $-P_f \delta V_f$. A small patch of boundary at height z , thickness dz and in-plane extent ds has initial vector area $d\mathbf{A} = dz ds \hat{\mathbf{n}}$. After deformation, this becomes $\text{Det}(F)F^{-T} \cdot d\mathbf{A}$. An incremental displacement $\delta\mathbf{u}$ displaces the patch by $(1 - 4z^2/a^2)\delta\mathbf{u}$ and hence changes the fluid volume by $-(1 - 4z^2/a^2)\delta\mathbf{u} \cdot \text{Det}(F)F^{-T} \cdot d\mathbf{A}$. Integrating this over the boundary gives $\delta V_f = -\oint \delta\mathbf{u} \cdot \int_{-a/2}^{a/2} (1 - 4z^2/a^2) \text{Det}(F)F^{-T} dz \cdot \hat{\mathbf{n}} \cdot ds$. Conducting the z integral then gives $\delta V_f = -\frac{2a}{3} \oint \delta\mathbf{u} \cdot \text{Det}(G)G^{-T} \cdot \hat{\mathbf{n}} \cdot ds$ and hence the appropriate boundary conditions are

$$(\mu G + (P_f - P)\text{Det}(G)G^{-T}) \cdot \hat{\mathbf{n}} = 0, \quad (6)$$

which, with eqs. (4), (5), specify the problem.

We first solve these equations for fingering in a simple Cartesian geometry to uncover the basic mechanisms at play, considering an elastic layer in an infinite strip with $0 < y < l$ and $-\infty < x < \infty$, an invading fluid at pressure P_f for $y < 0$ and a vacuum for $y > l$. We expect fingering of the $y = 0$ boundary at a critical P_f , so we write the fields as a translationally invariant base state plus a small perturbation:

$$\mathbf{u} = Y_1(y)\hat{\mathbf{y}} + \epsilon\mathbf{u}_2(\mathbf{x}, y), \quad P = P_1(y) + \epsilon P_2(x, y). \quad (7)$$

Substituting these into eqs. (4), (5) and setting $\epsilon = 0$, we see that Y_1 is a constant and P_1 is linear in y . Applying

eq. (6) at $y = 0$ and at $y = l$ (where $P_f = 0$) then yields

$$Y_1(y) = a^2 P_f / (8l\mu), \quad P_1(y) = \mu + P_f - P_f y / l. \quad (8)$$

Expanding eqs. (4)–(6) to $\mathcal{O}(\epsilon)$ around this state gives us an eigenvalue problem for the base-state's stability:

$$\frac{8\mu}{a^2} \mathbf{u}_2 = \frac{4\mu}{5} \nabla^2 \mathbf{u}_2 - \nabla P_2 + \frac{4}{5} \nabla P_1 \cdot (\nabla \mathbf{u}_2), \quad (9)$$

$$\nabla \cdot \mathbf{u}_2 = 0, \quad (10)$$

$$\left(\frac{4}{5} \mu \nabla \mathbf{u}_2 - P_2 + \frac{4}{5} \mu (\nabla \mathbf{u}_2)^T \right) \cdot \hat{\mathbf{n}} = 0. \quad (11)$$

Assuming explicit oscillatory perturbative fields, $P_2 = P_2(y) \cos(kx)$, $\mathbf{u}_2 = Y_2(y) \cos(kx) \hat{\mathbf{y}} + X_2(y) \sin(kx) \hat{\mathbf{x}}$, we solve these equations and see that, provided $l \gg a$, the boundary destabilizes when

$$P_f = \frac{2\mu l a^2 k^2 (ak (ak - \sqrt{a^2 k^2 + 10}) + 10) + 25}{5a ak}. \quad (12)$$

Minimizing this threshold over k , we see that fluid-driven fingering of a rectilinear elastic meniscus occurs with wavelength and pressure

$$\lambda \approx 2.75 \dots a, \quad P_f \approx 10.1 \dots l\mu/a. \quad (13)$$

We next consider the experimental circular geometry [18]. A naive extrapolation of our Cartesian stability analysis result to the circular case by taking $l \sim r_2 - r_1$ would predict threshold pressures far beyond those observed because the Cartesian base state is 1d whereas in the circular one is 2d, with different qualitative forms for the decay of the elastic fields. Assuming an annular elastic layer occupying the region $r_1 < r < r_2$, $-\pi < \theta < \pi$ with a fluid at pressure P_f in the cavity $r < r_1$ and a vacuum for $r > r_2$ allows us to write the displacement and pressure fields as

$$\mathbf{u} = R_1(r) \hat{\mathbf{r}} + \epsilon (R_2(r) \cos(n\theta) \hat{\mathbf{r}} + \Theta_2(r) \sin(n\theta) \hat{\boldsymbol{\theta}}), \quad (14)$$

$$P = P_1(r) + \epsilon P_2(r) \cos(n\theta). \quad (15)$$

Substituting these expressions into (4), (5), then setting $\epsilon = 0$, allows us to solve (5) for R_1 ,

$$R_1(r) = \frac{5r}{4} \left(\sqrt{1 + \left(\frac{c_4}{r} \right)^2} - 1 \right), \quad (16)$$

where the integration constant c_4 parameterizes the inner-boundary displacement. We can solve for P_1 analytically then solve the perturbative equations (9)–(11) numerically to find the fingering threshold and mode without further approximation (see appendix) but the algebra is cumbersome. However, the expressions simplify in the limit of thin layers, $a \ll r_1$, a case of much interest. As in the Cartesian geometry, we expect an instability when $R_1(r_1) \sim a$, when strains become geometrically large. Such displacements require $c_4 \sim \sqrt{r_1 a} \ll r_1$, so R_1 can be replaced by its first-order expansion $R_1(r) = 5c_4^2 / (8r)$. Furthermore, $R_1'(r) \sim c_4^2 / r^2$ is negligibly small so we can neglect gradients of \mathbf{u} , setting $G = I$. This reduces eq. (4) to $\frac{8\mu}{a^2} R_1(r) = -P_1'(r)$, which on integration yields $P_1 \sim \log(r)$. Similarly, applying the boundary

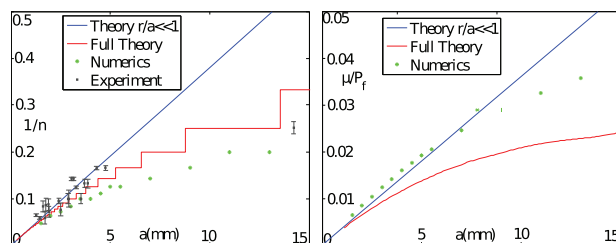


Fig. 2: (Colour on-line) A thin annular elastic layer with thickness a , shear modulus μ and radius $r_2 = 125$ mm has a central cavity of radius $r_1 = 11.5$ mm filled with a fluid at pressure P_f . Above a threshold P_f , fingers of fluid invades the layer. We show the inverse number of fingers $1/n$ (left) and the inverse scaled threshold pressure μ/P_f (right) as a function of the layer thickness. The plots compare the predictions of the full 2D theory based on eq. (16) (red lines), the asymptotic results for $a/r_1 \ll 1$ given in eq. (19) (blue lines), full finite-element results and, in the left plot, experimental results [18].

conditions (6) allows us to determine R_1 , P_1 as

$$R_1(r) = \frac{a^2 P_f}{8\mu r \log(r_2/r_1)}, \quad P_1(r) = \mu + \frac{P_f \log(r/r_2)}{\log(r_1/r_2)}. \quad (17)$$

Both fields only vary on length scales comparable to r_1 , so around the inner boundary ($|r - r_1| \ll r_1$) they are well described by their Taylor expansions around r_1 :

$$R_1 = \frac{a^2 P_f}{8\mu r_1 \log(r_2/r_1)}, \quad P_1 = \mu + P_f + \frac{P_f (r - r_1)}{r_1 \log(r_1/r_2)}. \quad (18)$$

Identifying $(r - r_1) \rightarrow y$ and $r_1 \log(r_2/r_1) \rightarrow l$, these results match the base state for the rectilinear case (eq. (8)). Fingering only occurs within a characteristic distance a from the boundary where base states match, so the instability will proceed in the same way with mode number ($n = 2\pi r_1 / \lambda$) and threshold

$$n \approx 2.28 r_1 / a, \quad P_f \approx 10.1 \mu (r_1 / a) \log(r_2 / r_1). \quad (19)$$

This pressure diverges logarithmically as $r_2 \rightarrow \infty$ so fingering will occur in a pressurized cavity in an almost infinite layer, but not in a wide rectilinear strip.

In fig. 2, we compare these predictions with experiments [18] and non-linear finite-element simulations carried out using a commercial package ABAQUS, using a pressure-based Lagrange multiplier method to enforce incompressibility to within 10^{-9} variations in the pressure, and see that the numerical results agree well with the theory and experiments for very thin layers. Our data extends to layers with $a/r_1 \gtrsim 1$ which are not thin; unsurprisingly, here the depth-averaged asymptotic theory predicts too few fingers and too high pressures. A better approximation, also shown in fig. 2, can be obtained by returning to the full expression for $R_1(r)$ (eq. (16)) without assuming $a \ll r_1$ (see appendix). The theory is still depth averaged so that it cannot capture the full behavior of thick layers, but it captures the qualitative nature of the non-linear deviations.

As alluded to in our introduction, fingering of a confined elastic layer can also be driven by transverse displacement [21,22]. Layer incompressibility implies that pulling the plates apart causes the meniscus to be inwardly displaced and, at a critical separation, fingers form in a manner reminiscent of fig. 1. The similarity arises despite the difference in the origin of the base states because both add volume to an incompressible layer, resulting in long-ranged displacements that only vary on in-plane length-scales. In the boundary region of characteristic width a where fingering occurs, both base states are essentially constant inward displacements, and finger identically. We now show how our theory makes this connection concrete. If the invading fluid is removed ($P_f = 0$) and instead the rigid plates are separated to $z = \pm(a + \Delta z)/2$, we must modify $\mathbf{V}(\mathbf{R})$ to

$$\mathbf{V}(\mathbf{R}) = (1 - 2z/a)(1 + 2z/a)\mathbf{u}(\mathbf{r}) + z\hat{\mathbf{z}}\Delta z/a. \quad (20)$$

Since separation adds volume to the whole layer area, while the inward displacement only does so at the boundary, for thin wide layers, the Δz required for displacement comparable to a will be small. Assuming $\Delta z/a \ll 1$, the above \mathbf{V} leads to the equations of equilibrium [22],

$$\frac{8\mu}{a^2}\mathbf{u} = \frac{4\mu}{5}\nabla^2\mathbf{u} - \text{Det}(G)G^{-T} \cdot \nabla P, \quad (21)$$

$$\text{Det}(G) = 1 - 6\Delta z/(5a), \quad (22)$$

$$(\mu G - P\text{Det}(G)G^{-T}) \cdot \hat{\mathbf{n}} = 0, \quad (23)$$

identical to the pressure-driven case except for the change of driving from P_f in the boundary condition to $6\Delta z/(5a)$ in eq. (22).

In the Cartesian strip geometry, we can solve eq. (22) for the translationally invariant displacement $Y_1(y) = \frac{3}{4}a(l\Delta z/a^2)(1 - 2(y/l))$, which is symmetric about $y = l/2$ and hence substantially different from the pressure-driven case. However, since this displacement only varies over distances comparable to l , in a region of width comparable to a around the $y = 0$ boundary, it is essentially constant, $Y_1(0) = \frac{3}{4}l\Delta z/a$. Substituting this constant into eqs. (21) and (23), we see that, in the same boundary region, the pressure is given by $P = \mu - 6\mu y\Delta z/a^3$. Thus, identifying $\Delta z \rightarrow a^3 P_f/(6l^2\mu)$, in this boundary region the separation-driven fields match the pressure-driven ones (eq. (8)), up to an offset P_f in the pressure.

We next consider the stability of these base states by considering small perturbations, $P = P_1(y) + \epsilon P_2(x, y)$ and $\mathbf{u} = \mathbf{u}_1(y) + \epsilon \mathbf{u}_2(x, y)$, localized to the $y = 0$ boundary. If we expand eqs. (21)–(23) to first order in ϵ , this is analogous to deriving eqs. (9)–(11). The only two differences are the offset in the base pressures by P_f , which simply cancels the offset by P_f between the two boundary conditions, and the $6\Delta z/(5a)$ term in eq. (22) which, in the thin-layer limit, is negligibly small. Thus, the stability of a thin layer is also governed by eqs. (9)–(11), and the instability proceeds in the same way, with threshold $\Delta z \approx 1.68a^2/l$. The same reasoning applies even with large perturbations, so the full non-linear finger development is identical.

In the annular geometry, we solve eq. (22) for the base state to get $R_1(r) = \frac{5r}{4} \left(\sqrt{1 - \frac{6\Delta z}{5a} + \left(\frac{c_4}{r}\right)^2} - 1 \right)$. As in the pressure-driven case, for thin layers with $a/r_1 \ll 1$, we may expand the root in the previous expression to get $R_1(r) = \frac{5c_4^2}{8r} - \frac{3\Delta zr}{4a}$. Solving eqs. (21)–(23) for the full base state then yields

$$P_1(r) = \mu + \frac{3\Delta z\mu}{a^3 \log(r_1/r_2)} \left(r^2 \log\left(\frac{r_1}{r_2}\right) + r_1^2 \log\left(\frac{r_2}{r}\right) + r_2^2 \log\left(\frac{r}{r_1}\right) \right), \quad (24)$$

$$R_1(r) = \frac{3\Delta z(r_1^2 - r_2^2)}{8ar \log(r_1/r_2)} - \frac{3\Delta zr}{4a}. \quad (25)$$

These fields vary on length scales comparable to $r_1 \gg a$, so in a region around the inner boundary with characteristic width a they are well approximated by their Taylor series around r_1 . Identifying

$$\Delta z \rightarrow \frac{P_f}{3\mu} \frac{a^3}{2r_1^2 \log(r_1/r_2) - r_1^2 + r_2^2}, \quad (26)$$

we see that the equivalent series differ from those in the pressure-driven case (eq. (18)) by the same offset of P_f to P_1 as in the Cartesian strip case. Thus, as before, the base states differ on long length scales but match around the inner boundary, and are susceptible to exactly the same fingering instability. Substituting the threshold pressure into the above expression for Δz , we find the threshold separation for fingering which, when $r_2 \gg r_1$, reduces to $\frac{\Delta z}{a} \approx 3.37 \frac{a}{r_2} \frac{r_1}{r_2} \log(r_2/r_1)$, and is indeed small.

We confirm this equivalence between fluid- and displacement-driven fingering via non-linear finite-element simulations using the commercial package ABAQUS, with incompressibility implemented using a pressure-based penalty method to an accuracy of 10^{-9} . Figure 3 shows the hysteresis loops and fingering patterns for the two cases. Despite the layers being only modestly thin ($r_1/a \sim 0.3$), the loops are very similar. The fingering transition is sub-critical in both cases, and hence both systems exhibit bistability.

Our study highlights the geometrical similarity and the essential physical differences between elastic and viscous fingering. Elastic fingering is governed by an equilibrium subcritical bifurcation that is reversible whilst viscous fingering is a rate-dependent dynamic process with a supercritical bifurcation driven by a competition between surface tension (γ) and viscous shear/pressure gradients. Furthermore, we also show that both displacement-controlled and pressure-controlled scenarios may be described by the same theory, contrary to what might be expected. Although we have ignored the effect of surface forces, surface tension will become important in elastic fingering if the layer thickness becomes comparable to the elastocapillary length scale γ/μ .

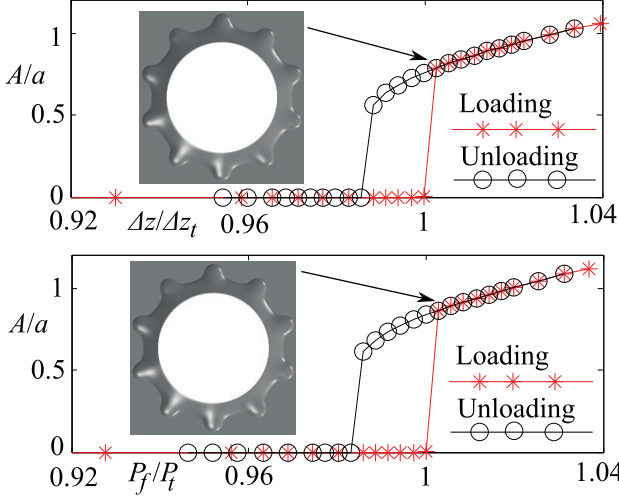


Fig. 3: (Colour on-line) Finite-element hysteresis loops showing finger amplitude A for displacement (top) and fluid pressure (bottom) driven fingering, using $a = 3.5$ mm, $r_1 = 11.5$ mm and $r_2 = 125$ mm. Both show a first-order transition to very similar fingered states (see insets) at threshold separation $\Delta z_t = 0.02a$ and pressure $P_t = 69.4\mu$, respectively. The dimensionless threshold ratio $(P_t/\mu)/(\Delta z_t/a) = 3510 \pm 10$ is close to the theoretical estimate of 3640 from eq. (26).

These may be relevant to many phenomena in adhesion science/engineering and perhaps even biological morphogenetic processes where branching and fingering abound.

We thank E. BOUCHAUD and B. SAINTYVES for introducing us to this experiment, for useful discussions and fig. 1(c), (d). We also thank Trinity Hall, Cambridge and the 1851 Royal Commission (JSB), the Harvard NSF-MRSEC DMR0820484 (ZW, LM) and the MacArthur Foundation (LM) for partial support.

APPENDIX

Here, we provide the details of the calculation for the fingering threshold and mode in an annular layer, continuing from eq. (16). In an (r, θ) circular polar coordinate system we note that,

$$\nabla P = \begin{pmatrix} P_{,r} \\ P_{,\theta} \\ P_{,r} \end{pmatrix}, \quad \nabla \mathbf{u} = \begin{pmatrix} u_{r,r} & \frac{u_{r,\theta} - u_{\theta}}{r} \\ u_{\theta,r} & \frac{u_{\theta,\theta} + u_r}{r} \end{pmatrix}, \quad (\text{A.1})$$

$$\nabla^2 \mathbf{u} = \begin{pmatrix} u_{r,rr} + \frac{u_{r,\theta\theta}}{r^2} + \frac{u_{r,r}}{r} - \frac{2u_{\theta,\theta}}{r^2} - \frac{u_r}{r^2} \\ u_{\theta,rr} + \frac{u_{\theta,\theta\theta}}{r^2} + \frac{u_{\theta,r}}{r} + \frac{2u_{r,\theta}}{r^2} - \frac{u_{\theta}}{r^2} \end{pmatrix}, \quad (\text{A.2})$$

where $A_{,b} = \partial A/\partial b$. We find the base pressure, $P_1(r)$, by considering the $\hat{\mathbf{r}}$ component of eq. (4) which, for a radial base-state displacement $R_1(r)\hat{\mathbf{r}}$, reduces to

$$\frac{8\mu}{a^2} R_1(r) = \frac{4\mu}{5} \left(\frac{rR_1'(r) - R_1(r)}{r^2} + R_1(r) \right) + \left(1 + \frac{4R_1(r)}{5r} \right) P_1'(r). \quad (\text{A.3})$$

Substituting in R_1 from eq. (16) and simplifying gives

$$a^2 r (c_4^2 + r^2)^2 P_1'(r) + \mu \left(a^2 c_4^4 + 10r^2 (c_4^2 + r^2) \left(-r\sqrt{c_4^2 + r^2} + c_4^2 + r^2 \right) \right) = 0 \quad (\text{A.4})$$

which we can solve to find $P_1(r)$ as

$$P_1 = P_0 - \mu \left(\frac{5r^2}{a^2} \left(1 - \sqrt{\frac{c_4^2}{r^2} + 1} \right) + \frac{c_4^2}{2(c_4^2 + r^2)} + \frac{5c_4^2}{a^2} \log \left(\sqrt{c_4^2 + r^2} + r \right) - \log \left(\sqrt{\left(\frac{c_4}{r} \right)^2 + 1} \right) \right), \quad (\text{A.5})$$

where P_0 is a constant of integration. Applying the boundary condition, eq. (6), on the inner and outer radii, gives

$$\mu \left(1 + \frac{4}{5} R_1'(r_1) \right) + (P_f - P_1(r_1)) \left(1 + \frac{4R_1(r_1)}{5r_1} \right) = 0, \quad (\text{A.6})$$

$$\mu \left(1 + \frac{4}{5} R_1'(r_2) \right) - P_1(r_2) \left(1 + \frac{4R_1(r_2)}{5r_2} \right) = 0. \quad (\text{A.7})$$

For algebraic convenience, instead of solving these for the integration constant, P_0 and c_4 , in terms of the applied pressure P_f , we solve them for P_f and P_0 in terms of c_4 :

$$\begin{aligned} \frac{P_0}{\mu} &= \frac{5r_2^2}{a^2} \left(1 - \sqrt{\frac{c_4^2}{r_2^2} + 1} \right) + \frac{5c_4^2}{a^2} \log \left(\sqrt{c_4^2 + r_2^2} + r_2 \right) \\ &\quad - \frac{c_4^2}{2(c_4^2 + r_2^2)} - \frac{1}{2} \log \left(\frac{c_4^2}{r_2^2} + 1 \right) + 1, \quad (\text{A.8}) \\ \frac{2P_f}{\mu} &= \frac{10c_4^2}{a^2} \log \left(\frac{\sqrt{c_4^2 + r_2^2} + r_2}{\sqrt{c_4^2 + r_1^2} + r_1} \right) \\ &\quad + 10 \left(\frac{r_1^2}{a^2} \left(\sqrt{1 + \left(\frac{c_4}{r_1} \right)^2} - 1 \right) - \frac{r_2^2}{a^2} \left(\sqrt{1 + \left(\frac{c_4}{r_2} \right)^2} - 1 \right) \right) \\ &\quad - \frac{r_1^2}{c_4^2 + r_1^2} + \log \left(\frac{c_4^2}{r_1^2} + 1 \right) + \frac{r_2^2}{c_4^2 + r_2^2} - \log \left(\frac{c_4^2}{r_2^2} + 1 \right). \quad (\text{A.9}) \end{aligned}$$

This fully specifies the base state. If we have a generic base state \mathbf{u}_1 and P_1 giving rise to an effective deformation gradient G_1 , and we add small perturbations

$$\mathbf{u} = \mathbf{u}_1 + \epsilon \mathbf{u}_2, \quad P = P_1 + \epsilon P_2, \quad (\text{A.10})$$

then expanding eqs. (4)–(6) about the base state to first order in ϵ yields

$$\frac{8\mu}{a^2} \mathbf{u}_2 = \frac{4\mu}{5} \nabla^2 \mathbf{u}_2 - \nabla P_2 \cdot \text{adj}(G_1) - \frac{4}{5} \nabla P_1 \cdot \text{adj}(\nabla \mathbf{u}_2), \quad (\text{A.11})$$

$$\text{Tr}(\text{adj}(G_1) \cdot \nabla \mathbf{u}_2) = 0, \quad (\text{A.12})$$

subject to the boundary conditions

$$\left(\mu \frac{4}{5} \nabla \mathbf{u}_2 - P_2 \text{adj}(G_1)^T + \frac{4}{5} (P_f - P_1) \text{adj}(\nabla \mathbf{u}_2)^T \right) \cdot \hat{\mathbf{n}} = 0, \quad (\text{A.13})$$

where adj denotes the adjugate matrix.

Linearizing about the circular base state, we assume oscillatory forms for the perturbations

$$\mathbf{u} = R_1(r) \hat{\mathbf{r}} + \epsilon (R_2(r) \cos(n\theta) \hat{\mathbf{r}} + \Theta_2(r) \sin(n\theta) \hat{\boldsymbol{\theta}}), \quad (\text{A.14})$$

$$P = P_1(r) + \epsilon P_2(r) \cos(n\theta), \quad (\text{A.15})$$

from which we get

$$\nabla \mathbf{u}_2 = \begin{pmatrix} R_2'(r) \cos(n\theta) & -\frac{nR_2(r) + \Theta_2(r)}{r} \sin(n\theta) \\ \Theta_2'(r) \sin(n\theta) & \frac{R_2(r) + n\Theta_2(r)}{r} \cos(n\theta) \end{pmatrix}, \quad (\text{A.16})$$

$$\nabla P_2 = \begin{pmatrix} P_2'(r) \cos(n\theta) \\ -nP_2(r) \sin(n\theta) \end{pmatrix}, \quad (\text{A.17})$$

so, the θ component of eq. (A.11) is an algebraic equation for $P_2(r)$ with the solution

$$P_2(r) = \frac{4(\Theta_2(r)(\mu(a^2(n^2+1)+10r^2)+a^2rP_1'(r)))}{a^2nr(4R_1'(r)+5)} + \frac{4(a^2nR_2(r)(2\mu+rP_1'(r))-a^2\mu r(r\Theta_2''(r)+\Theta_2'(r)))}{a^2nr(4R_1'(r)+5)} \quad (\text{A.18})$$

and, similarly, since G_1 is diagonal, eq. (A.12) is an algebraic equation for Θ_2 with the solution

$$\begin{aligned} \Theta_2(r) &= -\frac{(4R_1(r)+5r)R_2'(r)}{n(4R_1'(r)+5)} - \frac{R_2(r)}{n} \\ &= -\frac{(c_4^2+r^2)R_2'(r)+rR_2(r)}{nr}. \end{aligned} \quad (\text{A.19})$$

The r component of eq. (A.11) yields an ode for $R_2(r)$:

$$\begin{aligned} &a^2(4R_2(r)(\mu+\mu n^2+rP_1'(r))+4n\Theta_2(r)(2\mu+rP_1'(r))) \\ &+r((4R_1(r)+5r)P_2'(r)-4\mu(rR_2''(r)+R_2'(r))) \\ &+40\mu r^2R_2(r) = 0. \end{aligned} \quad (\text{A.20})$$

Substituting in the above forms for R_1 , P_1 , Θ_2 and P_2 yields a non-linear fourth-order differential equation. It is accompanied by the four boundary conditions given by applying eq. (A.13) on the inner and outer radii:

$$\begin{aligned} &4(P_f - P_1(r_1))(n\Theta_2(r_1) + R_2(r_1)) \\ &-P_2(r_1)(4R_1(r_1) + 5r_1) + 4\mu r_1 R_2'(r_1) = 0, \end{aligned} \quad (\text{A.21})$$

$$(P_f - P_1(r_1))(nR_2(r_1) + \Theta_2(r_1)) + \mu r_1 \Theta_2'(r_1) = 0, \quad (\text{A.22})$$

$$\begin{aligned} &4P_1(r_2)(n\Theta_2(r_2) + R_2(r_2)) \\ &+P_2(r_2)(4R_1(r_2) + 5r_2) - 4\mu r_2 R_2'(r_2) = 0, \end{aligned} \quad (\text{A.23})$$

$$P_1(r_2)(nR_2(r_2) + \Theta_2(r_2)) - \mu r_2 \Theta_2'(r_2) = 0. \quad (\text{A.24})$$

We solve the system (A.20)–(A.24) using the Matlab's `bvp4c` boundary value solver by specifying values for n , r_1 , r_2 and a to find the lowest value of c_4 for which the equations have a solution, and find the solution. We then iterate over n until we find the solution with the lowest value of c_4 (that is the lowest displacement on the inner boundary), to find the first unstable mode, which sets the threshold and mode-number for fingering. We finally use eq. (A.9) to recover the fluid pressure threshold from the value of c_4 . The threshold and mode-number predictions from this procedure are shown in fig. 2 in our main text.

REFERENCES

- [1] SAFFMAN P. and TAYLOR G., *Proc. R. Soc. Lond. A*, **245** (1958) 312.
- [2] BATAILLE J., *Rev. Inst. Pét.*, **23** (1968) 1349.
- [3] NITTMAN J., DACCORD G. and STANLEY M., *Nature*, **314** (1985) 391.
- [4] PARK S. and DURIAN D., *Phys. Rev. Lett.*, **72** (1994) 3347.
- [5] AMAR M. B. and POIRÉ E. C., *Phys. Fluids*, **11** (1999) 1757.
- [6] LINDNER A., BONN D., POIRÉ E. C., AMAR M. B. and MEUNIER J., *J. Fluid Mech.*, **469** (2002) 237.
- [7] KONDIC L., SHELLEY M. J. and PALFFY-MUHORAY P., *Phys. Rev. Lett.*, **80** (1998) 1433.
- [8] KAWAGUCHI M., *Nonlinear Anal.: Theory, Methods Appl.*, **47** (2001) 907.
- [9] COUSSOT P., *J. Fluid Mech.*, **380** (1999) 363.
- [10] LINDNER A., COUSSOT P. and BONN D., *Phys. Rev. Lett.*, **85** (2000) 314.
- [11] MORA S. and MANNA M., *Phys. Rev. E*, **80** (2009) 016308.
- [12] HIRATA T., *Phys. Rev. E*, **57** (1998) 1772.
- [13] LEMAIRE E., LEVITZ P., DACCORD G. and VAN DAMME H., *Phys. Rev. Lett.*, **67** (1991) 2009.
- [14] WILSON S., *J. Fluid Mech.*, **220** (1990) 413.
- [15] MORA S. and MANNA M., *J. Non-Newtonian Fluid Mech.*, **173** (2012) 30.
- [16] MORA S. and MANNA M., *Phys. Rev. E*, **81** (2010) 026305.
- [17] NASE J., LINDNER A. and CRETON C., *Phys. Rev. Lett.*, **101** (2008) 074503.
- [18] SAINTYVES B., DAUCHOT O. and BOUCHAUD E., *Phys. Rev. Lett.*, **111** (2013) 047801.
- [19] GHATAK A., CHAUDHURY M., SHENOY V. and SHARMA A., *Phys. Rev. Lett.*, **85** (2000) 4329.
- [20] ADDA-BEDIA M. and MAHADEVAN L., *Proc. R. Soc. Lond. A*, **462** (2006) 3233.
- [21] SHULL K., FLANIGAN C. and CROSBY A., *Phys. Rev. Lett.*, **84** (2000) 3057.
- [22] BIGGINS J. S., SAINTYVES B., WEI Z., BOUCHAUD E. and MAHADEVAN L., *Proc. Natl. Acad. Sci. U.S.A.*, **110** (2013) 12545.